

# Druckminderventil, federbelastet Pressure-Reducing-Valve, springloaded

für Dämpfe, Gase und Flüssigkeiten, für sehr kleinen Minderdruck  
for steam, gases and liquids, for very low reduced pressure

## Typ 75

### Discharge capacities for saturated steam

for definition the size of Pressure-Reducing-Valve

Size		0	I	II		III		III B		
Over- pressure $p_d$ [bar(g)]	Nominal pipe	10	15	20	25	32	40	50	65	80
		3/8	1/2	3/4	1	1¼	1½	2	2½	3
		kg/h								
t <sub>max</sub> 200 °C	0,15	4	10	17	27	40	83	120	180	260
		5	11	19	31	46	99	145	210	310
		6	13	23	35	55	112	160	240	360
		7	16	28	46	70	140	200	300	440
		9	20	35	57	85	175	250	370	560
	1	11	25	42	68	100	210	300	450	680
		14	32	55	90	140	280	400	590	880
		17	40	70	115	170	350	520	750	1120
	2,5	21	47	84	135	200	400	600	880	1310
		24	55	99	155	240	480	700	1020	1540
	4	31	70	123	195	300	600	890	1300	1900
		38	85	150	245	360	740	1080	1600	2400
		46	104	185	300	450	900	1340	1950	2900
		54	122	225	350	540	1100	1600	2400	3400
		62	140	250	400	600	1250	1800	2700	4000

a) To the definition of the right valve size according to the table, the downstream pressure is considerably. The usual piping speeds are appropriate for the table codes.

b) The valve size determined under a) can be selected around a nominal size smaller, if it is noted that the pipe diameter at the valve outlet is increased around at least one nominal size.

To small pressure ratios applies:

$$\frac{\text{absolute reduced pressure } p \text{ [bar]}}{\text{absolute inlet pressure } p \text{ [bar]}} \begin{cases} \geq 0,7 \Rightarrow \text{correction factor} = 1,25 \\ \geq 0,8 \Rightarrow \text{correction factor} = 1,60 \\ \geq 0,9 \Rightarrow \text{correction factor} = 2,25 \end{cases}$$

$$\dot{m}_D = \dot{m}_D^1 \cdot f$$

The found correction factor must be multiplied due to the smaller flow rate by the given mass flow. With the help of the calculated value now a valve can be determined in accordance with the table.

With smaller pressure ratios than 0.7 no correction factor is used.

For superheated steam:

$$\dot{m}_D = \frac{V_H}{V_S} \cdot \dot{m}_D^1 \cdot f$$

If the downstream pipe should be longer than 3 meters, then it is to be selected around one nominal size stage more largely.

Gaskets for steam:

$P_1 < 4$  [bar(g)] (<150°C): Piston gasket PTFE  
Gasket ring EPDM

$P_1 < 15$  [bar(g)] (<200°C): Piston gasket PTFE  
Gasket ring AF 100

\*  $V_H$ : specific volume of the superheated steam

\*  $V_S$ : specific volume of the saturated steam

f : correction factor

$\dot{m}_D^1$ : given mass flow

$\dot{m}_D$ : resulting value of the mass flow, with that the table can be used.

\* see VDI Steam table